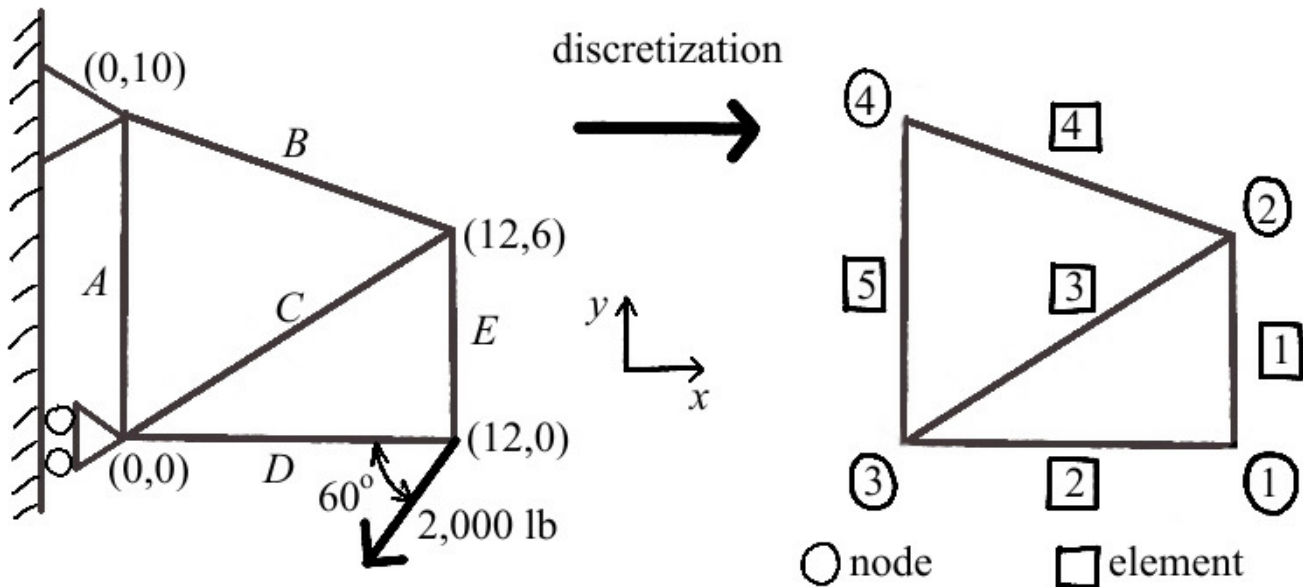


MCG 4102 / 5108 - Assignment 1 Solutions

Using the truss example given in class (4 nodes, and 5 elements involving 2 materials):



1) For element 1 in its local coordinate system, $[K^{(1)'}] = \frac{A^{(1)}E^{(1)}}{L^{(1)}} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$.

Material flag set to 1: 0.50-in steel: $A^{(1)} = \frac{\pi}{4}(0.5)^2 = 0.196 \text{ in}^2$

$E^{(1)} = 30 \times 10^6 \text{ psi}$

$L^{(1)} = \sqrt{(12-12)^2 + (6-0)^2} = 6 \text{ in}$

so that $\frac{A^{(1)}E^{(1)}}{L^{(1)}} = 10^3 \times 982 \text{ lbf/in.}$

In the global coordinate system, $[K^{(1)}] = \frac{A^{(1)}E^{(1)}}{L^{(1)}} \begin{bmatrix} n_{11}^2 & n_{21}n_{11} & -n_{11}^2 & -n_{21}n_{11} \\ n_{11}n_{21} & n_{21}^2 & -n_{11}n_{21} & -n_{21}^2 \\ -n_{11}^2 & -n_{21}n_{11} & n_{11}^2 & n_{21}n_{11} \\ -n_{11}n_{21} & -n_{21}^2 & n_{11}n_{21} & n_{21}^2 \end{bmatrix}$, with

$n_{11} = \frac{x_j - x_i}{L^{(1)}} = \frac{12-12}{6} = 0$. Finally, $[K^{(1)}] = 10^3 \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 982 & 0 & -982 \\ 0 & 0 & 0 & 0 \\ 0 & -982 & 0 & 982 \end{bmatrix} \text{ lbf/in.}$

2) Let us first create a null assemblage stiffness matrix $[K^a]$ involving global node numbers 1 to 4, with 2 degrees of freedom per node.

$$[K^a] = \begin{matrix} & \begin{matrix} 1 & 2 & 3 & 4 \end{matrix} \\ \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \end{matrix} & \begin{bmatrix} 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} \\ 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} \\ 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} \\ 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} & 0_{2 \times 2} \end{bmatrix} \end{matrix}$$

At node 1, stiffness contribution from elements 1 and 2.
 At node 2, stiffness contribution from elements 1, 3 and 4.
 At node 3, stiffness contribution from elements 2, 3 and 5.
 At node 4, stiffness contribution from elements 4 and 5. Then,

$$[K^a] = \begin{matrix} & \begin{matrix} 1 & 2 & 3 & 4 \end{matrix} \\ \begin{matrix} 1 \\ 2 \\ 3 \\ 4 \end{matrix} & \begin{bmatrix} K_{1,1}^{(1)} + K_{1,1}^{(2)} & K_{1,2}^{(1)} & K_{1,3}^{(2)} & 0_{2 \times 2} \\ K_{2,1}^{(1)} & K_{2,2}^{(1)} + K_{2,2}^{(3)} + K_{2,2}^{(4)} & K_{2,3}^{(3)} & K_{2,4}^{(4)} \\ K_{3,1}^{(2)} & K_{3,2}^{(3)} & K_{3,3}^{(2)} + K_{3,3}^{(3)} + K_{3,3}^{(5)} & K_{3,4}^{(5)} \\ 0_{2 \times 2} & K_{4,2}^{(4)} & K_{4,3}^{(5)} & K_{4,4}^{(4)} + K_{4,4}^{(5)} \end{bmatrix} \end{matrix}$$

$$\text{Given } [K^{(1)}] = 10^3 \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 982 & 0 & -982 \\ 0 & 0 & 0 & 0 \\ 0 & -982 & 0 & 982 \end{bmatrix} \text{ lbf/in, } [K^{(2)}] = 10^3 \begin{bmatrix} 115 & 0 & -115 & 0 \\ 0 & 0 & 0 & 0 \\ -115 & 0 & 115 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \text{ lbf/in,}$$

$$[K^{(3)}] = 10^3 \begin{bmatrix} 351 & 176 & -351 & -176 \\ 176 & 88 & -176 & -88 \\ -351 & -176 & 351 & 176 \\ -176 & -88 & 176 & 88 \end{bmatrix} \text{ lbf/in, } [K^{(4)}] = 10^3 \begin{bmatrix} 98 & -33 & -98 & 33 \\ -33 & 11 & 33 & -11 \\ -98 & 33 & 98 & -33 \\ 33 & -11 & -33 & 11 \end{bmatrix} \text{ lbf/in}$$

$$\text{and } [K^{(5)}] = 10^3 \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 589 & 0 & -589 \\ 0 & 0 & 0 & 0 \\ 0 & -589 & 0 & 589 \end{bmatrix} \text{ lbf/in, we obtain}$$

$$[K^a] = 10^3 \begin{bmatrix} 115 & 0 & 0 & 0 & -115 & 0 & 0 & 0 \\ & 982 & 0 & -982 & 0 & 0 & 0 & 0 \\ & & 450 & 143 & -351 & -176 & -98 & 33 \\ & & & 1080 & -176 & -88 & 33 & -11 \\ & & & & 466 & 176 & 0 & 0 \\ & & & & & 677 & 0 & -589 \\ & & & & & & 98 & -33 \\ \text{Sym} & & & & & & & 600 \end{bmatrix} \text{ lbf/in}$$

3) Because of boundary conditions $u_3 = u_4 = v_4 = 0$, the rows and columns corresponding to these degrees of freedom can be removed from $[K^a]$. We are left with

$$[K]\{U\} = \{F\} \text{ or } 10^3 \begin{bmatrix} 115 & 0 & 0 & 0 & 0 \\ & 982 & 0 & -982 & 0 \\ & & 450 & 143 & -176 \\ & & & 1080 & -88 \\ \text{Sym} & & & & 677 \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ v_3 \end{Bmatrix} = \begin{Bmatrix} -1,000 \\ -1,732 \\ 0 \\ 0 \\ 0 \end{Bmatrix}.$$

4) $\{U\} = [K]^{-1}\{F\}$, which yields $u_1 = -0.00879 \text{ in}$ $u_2 = 0.01012 \text{ in}$
 $v_1 = -0.03581 \text{ in}$ $v_2 = -0.03405 \text{ in}$
 $u_3 = 0$ $u_4 = 0$
 $v_3 = -0.00180 \text{ in}$ $v_4 = 0$

5) In element 3, by definition, the axial extension $\delta = u'_j - u'_i = u'_2 - u'_3$ with $u'_2 = n_{11}u_2 + n_{21}v_2$ and $u'_3 = n_{11}u_3 + n_{21}v_3$. For element 3, $n_{11} = \frac{x_2 - x_3}{L^{(3)}} = \frac{12 - 0}{13.42} = 0.8944$, and $n_{21} = \frac{y_2 - y_3}{L^{(3)}} = \frac{6 - 0}{13.42} = 0.4472$.

Then, $\delta = -0.0057 \text{ in}$. Strain $\varepsilon = \frac{\delta}{L^{(3)}}$, with $L^{(3)} = \sqrt{(0 - 12)^2 + (0 - 6)^2} = 13.42 \text{ in}$, from which $\varepsilon = -0.4 \times 10^{-3} \text{ in/in}$. Stress $\sigma = E\varepsilon$, with $E^{(3)} = 30 \times 10^6 \text{ psi}$, from which $\sigma = -12,004 \text{ psi}$ (compression). Finally, $F = \sigma A^{(3)}$, with $A^{(3)} = \frac{\pi}{4}(0.5)^2 = 0.196 \text{ in}^2$, from which $F^{(3)} = -2352 \text{ lbf}$ (compressive force).

Note: Numerical values may change slightly due to round-off errors.