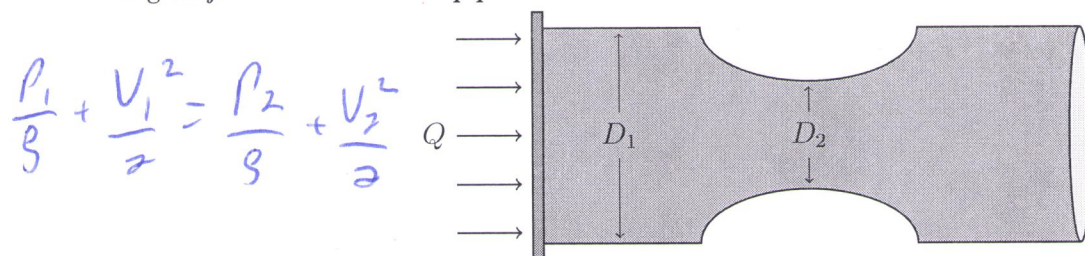


1) a) List four assumptions necessary to apply Bernoulli's equation for either compressible or incompressible flow.

- ① - steady state
- ① - inviscid ( $\mu=0$ )
- ① - along streamline
- ① - no other sources/sinks of energy

b) Water enters a pipe of diameter,  $D_1$  at a relative pressure  $P_1$ . The pipe contains a contraction to a diameter,  $D_2$ . What is the maximum flow rate,  $Q$ , possible such that the water will not cavitate in the throat of the contraction. The water will cavitate if its pressure drops below its saturation pressure at a given temperature,  $P_{\text{sat}}$ .

Neglect friction losses in the pipe.



$$\textcircled{1} \quad V_1 = \frac{Q}{A_1} = \frac{4Q}{\pi D_1^2} \quad V_2 = \frac{Q}{A_2} = \frac{4Q}{\pi D_2^2} \quad P_{\text{abs}} = P_1 + P_{\text{atm}} \quad P_{\text{abs}} = P_{\text{sat}} \quad \textcircled{1}$$

$$\frac{P_1 + P_{\text{atm}}}{\rho} + \frac{1}{2} \left( \frac{4Q}{\pi D_1^2} \right)^2 = \frac{P_{\text{sat}}}{\rho} + \frac{1}{2} \left( \frac{4Q}{\pi D_2^2} \right)^2$$

$$\textcircled{1} \quad \frac{P_1 + P_{\text{atm}} - P_{\text{sat}}}{\rho} = \frac{1}{2} \left( \frac{16Q^2}{\pi^2} \right) \left( \frac{1}{D_2^4} - \frac{1}{D_1^4} \right) = \frac{1}{2} \left( \frac{16Q^2}{\pi^2} \right) \left( \frac{D_1^4 - D_2^4}{D_2^4 D_1^4} \right)$$

$$Q^2 = \frac{(P_1 + P_{\text{atm}} - P_{\text{sat}}) D_1^4 D_2^4 \pi^2}{8 \rho (D_1^4 - D_2^4)}$$

$$\textcircled{1} \quad Q = \frac{D_1^2 D_2^2 \pi}{2} \sqrt{\frac{P_1 + P_{\text{atm}} - P_{\text{sat}}}{2 \rho (D_1^4 - D_2^4)}}$$

- 2) Find the force required to hold a cylindrical plug of diameter,  $D_2$ , in place at the end of a circular water pipe of diameter,  $D_1$ , as a function of flow rate,  $Q$ .

Assume frictionless flow and neglect gravitational effects.

$$A_1 = \frac{\pi D_1^2}{4}$$

$$A_2 = \frac{\pi(D_1^2 - D_2^2)}{4}$$

$$A_1 - A_2 = \frac{\pi D_2^2}{4}$$

$$P_2 = P_{atm}$$

$$V_1 = \frac{Q}{A_1}$$

$$V_2 = \frac{Q}{A_2} \quad \textcircled{1}$$

$$P_1 + \frac{\rho V_1^2}{2} = P_2 + \frac{\rho V_2^2}{2} \Rightarrow P_1 = \frac{\rho}{2} (V_2^2 - V_1^2) \quad \textcircled{1}$$

$$\textcircled{1} \quad \sum F_x = \sum v_x \rho \vec{V} \cdot \hat{n} A$$

$$\textcircled{1} \quad -F + F_p = -V_1^2 \rho A_1 + V_2^2 \rho A_2$$

$$\downarrow \quad F_p = \frac{\rho}{2} (V_2^2 - V_1^2) A_1$$

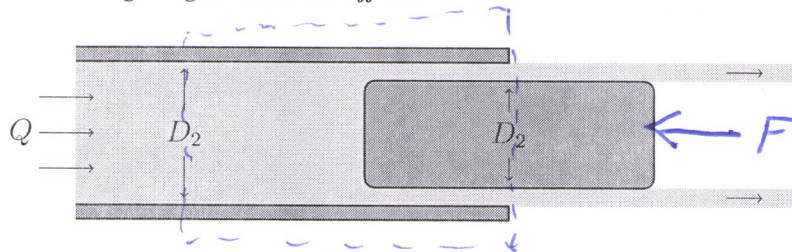
$$\textcircled{1} \quad F = F_p + V_1^2 \rho A_1 - V_2^2 \rho A_2$$

$$= \rho Q^2 \left\{ \left[ \frac{1}{2A_2^2} - \frac{1}{2A_1^2} \right] A_1 + \frac{1}{A_1} - \frac{1}{A_2} \right\}$$

$$= \rho Q^2 \left\{ \frac{A_1}{2A_2^2} - \frac{1}{2A_1} + \frac{1}{A_1} - \frac{1}{A_2} \right\}$$

$$= \rho Q^2 \left\{ \frac{A_1^2 - 2A_1 A_2 + A_2^2}{2A_2^2 A_1} \right\}$$

$$= \rho Q^2 \left\{ \frac{(A_1 - A_2)^2}{2A_1 A_2^2} \right\}$$



$$F = \frac{\rho Q^2}{2} \left\{ \frac{\left( \frac{\pi}{4} D_2^2 \right)^2}{\frac{\pi}{4} D_1^2 \left( \frac{\pi}{4} [D_1^2 - D_2^2] \right)^2} \right\}$$

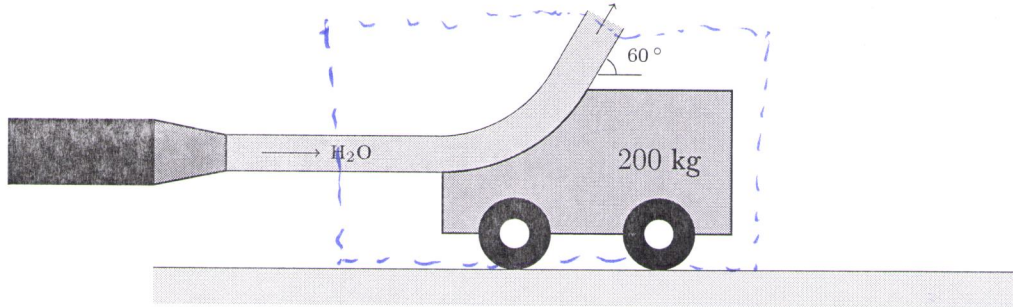
$$\textcircled{1} \quad F = \frac{2\rho Q^2}{\pi} \left( \frac{D_2^4}{D_1^2 (D_1^2 - D_2^2)^2} \right)$$

in the negative x direction.

Name: \_\_\_\_\_

3) A jet of water with speed,  $V_j = 20$  m/s hits a 200 kg cart and is turned by  $60^\circ$ . The jet maintains a cross-sectional area,  $S_j = 0,02$  m<sup>2</sup>. At time,  $t = 0$  s, the cart is at rest at a position,  $x = 0$  m. Find an expression for the speed of the cart for  $t > 0$  s.

You can neglect friction in the wheels, variation in the momentum of the water that is in contact with the cart, and gravitational effects.



$$\sum \vec{F}_x - \int_{c.v.} a_x \rho dV = \frac{\partial}{\partial t} \int_{c.v.} u_x \rho dV + \int_{c.s.} u_x \rho \vec{v} \cdot \vec{n} ds$$

$$\textcircled{1} \quad -\frac{dU}{dt} M = \int_{c.s.} u_x \rho \vec{v} \cdot \vec{n} ds$$

$$\begin{aligned} \vec{v}_1 &= (V_j - u) \hat{i} \\ \vec{v}_2 &= (V_j - u) [\cos\theta \hat{i} + \sin\theta \hat{j}] \end{aligned} \quad \textcircled{1}$$

$$\begin{aligned} \textcircled{1} \quad -\frac{dU}{dt} M &= -V_1 \times \rho |\vec{v}_1| S + V_2 \times \rho |\vec{v}_2| S \\ &= -V_1^2 \rho S + V_2^2 \rho S \cos\theta \\ &= -\frac{1}{2} V_1^2 \rho S \end{aligned}$$

$$\frac{V_j \rho S}{2M} = \frac{(20 \text{ m/s}) (1000 \text{ kg/m}^3) (0,02 \text{ m}^2)}{2 (200 \text{ kg})} = 1 \text{ s}^{-1}$$

$$\frac{dU}{dt} = \frac{\rho S}{2M} (V_j - u)^2$$

$$\textcircled{1} \quad \int_0^u \frac{d\hat{u}}{(V_j - \hat{u})^2} = \int_0^{\hat{t}} \frac{\rho S}{2M} d\hat{t}$$

$$\int_0^u \frac{-d(V_j - \hat{u})}{(V_j - \hat{u})^2} = \left[ \frac{1}{V_j - \hat{u}} \right]_0^u = \frac{1}{V_j - u} - \frac{1}{V_j} = \frac{\rho S}{2M} \hat{t} \Rightarrow$$

$$u = V_j \left[ \frac{\frac{V_j \rho S}{2M} \hat{t}}{1 + \frac{V_j \rho S}{2M} \hat{t}} \right]$$

$$\textcircled{1} \quad u = 20 \text{ m/s} \left[ \frac{\hat{t}}{1 + \hat{t}} \right]$$