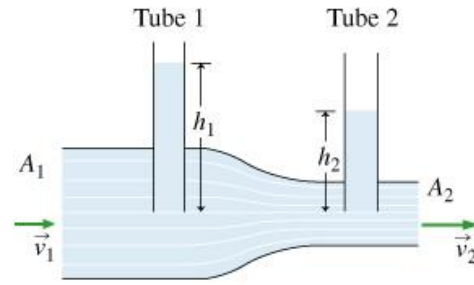


Problem 1 (3 points)

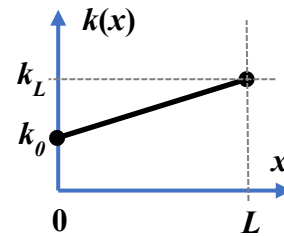
Consider a pair of vertical, open-ended glass tubes inserted into a horizontal pipe with a horizontal pipe carrying a steadily moving, incompressible fluid of density ρ (see figure) flowing with negligible dissipation. The cross-sectional area of the pipe is A_1 at the position of *tube 1*, and A_2 at the position of *tube 2*. The fluid rises to heights h_1 and h_2 in the two open-ended tubes.



a) With g the magnitude of the acceleration due to gravity (take $g \sim 10 \text{ m s}^{-2}$), $h_1 = 200 \text{ mm}$, $h_2 = 100 \text{ mm}$, $A_1 = 1 \text{ mm}^2$, and $A_2 = 0.5 \text{ mm}^2$, calculate v_1 , the speed of the fluid in the left end of the main pipe. Express the result in the SI units.

Problem 2 (5 points)

Find the thermal resistance $R = (T_H - T_C)/H$ of a rod of constant section A and total length L when the thermal conductivity of the material forming the rod changes linearly from $x = 0$ to $x = L$ according to the graph showed to the right. (T_H is the temperature of the hot end of the rod and T_C is the temperature of the cold end of the rod.)



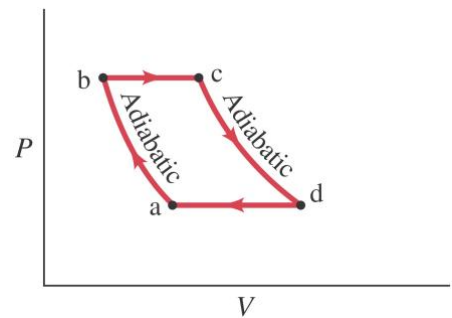
(Hint: You might find useful the following integral $\int x^{-1} dx = \ln x + \text{constant}$.)

Problem 3 (5 points)

The thermodynamic cycle showed in figure makes an ideal gas undergoing two adiabatic transformations ($a \rightarrow b$ and $c \rightarrow d$) and two isobaric transformations ($b \rightarrow c$ and $d \rightarrow a$). Show that the efficiency of this cycle is

$$e = 1 - \left(\frac{P_b}{P_a}\right)^{\frac{1-\gamma}{\gamma}}$$

where $\gamma = C_P/C_V$.



Problem 4 (5 points)

A cup of mass 200 g initially at temperature 20 °C is filled with 400 g of a liquid initially at temperature 100 °C. The cup has a specific heat $c_{\text{cup}} = 1000 \text{ J/(Kg } ^\circ\text{C)}$ and the liquid has a specific heat $c_{\text{liq}} = 5000 \text{ J/(Kg } ^\circ\text{C)}$.

Calculate the final temperature (T_{final}) at the thermal equilibrium and the total change in entropy (ΔS) as a result of the process leading to the thermal equilibrium.

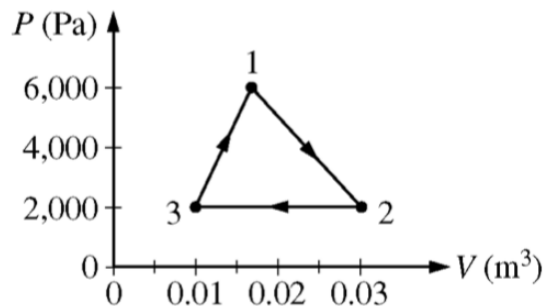
Problem 5 (1 points)

Which of the following is true about any system that undergoes a reversible thermodynamic process?

- (A) There are no changes in the internal energy of the system.
- (B) The temperature of the system remains constant during the process.
- (C) The entropy of the system and its environment remains unchanged.
- (D) The entropy of the system and its environment must increase.
- (E) The net work done by the system is zero.
- (F) None of the above

Problem 6 (1 points)

A sample of nitrogen gas (N_2) undergoes the cyclic thermodynamic process shown here. Which of the following gives the net heat transferred to the system in one complete cycle $1 \rightarrow 2 \rightarrow 3 \rightarrow 1$?



- (A) - 80 J
- (B) - 40 J
- (C) + 40 J
- (D) +80 J
- (E) +180 J
- (F) None of the above

Solutions

Problem 1 (3 points)

- This is the solved example in the book 12.9.
- It was also one of the homework problems that had 95% of correct answers.

Solutions:

The pressure in zone-1 is $p_1 = \rho g h_1$ and using the Bernoulli equation plus the continuity equation we obtain:

$$\sqrt{\frac{2g(h_1 - h_2)}{\left(\frac{A_1}{A_2}\right)^2 - 1}}$$

Then

$$v_1 = \sqrt{(2/3)} \text{ m/s}$$

Problem 2 (5 points)

Solutions:

Given the linear dependence in the graph,
 $\kappa(x)$ follows the law

$$\kappa(x) = \kappa_0 + (\kappa_L - \kappa_0) \frac{x}{L}$$

The heat current is given by

$$H = -\kappa(x) A \frac{dT}{dx}$$

then

$$-\frac{1}{H} \int dT = \frac{1}{A} \int \frac{dx}{\kappa(x)}$$

$$-\frac{1}{H} \int_{T_0=T_H}^{T_2=T_C} dT = \frac{T_H - T_C}{H} = R ; \quad \frac{1}{A} \int_0^L \frac{dx}{\kappa_0 + (\kappa_L - \kappa_0) \frac{x}{L}}$$

Introducing the variable $\kappa = \kappa_0 + (\kappa_L - \kappa_0) \frac{x}{L}$
and then $d\kappa = \frac{\kappa_L - \kappa_0}{L} dx$

$$R = \frac{1}{A} \frac{L}{\kappa_L - \kappa_0} \int_{\kappa_0}^{\kappa_L} \frac{d\kappa}{\kappa} = \frac{1}{A} \frac{L}{\kappa_L - \kappa_0} \ln \kappa \Big|_{\kappa_0}^{\kappa_L}$$

$$R = \frac{L}{A(\kappa_L - \kappa_0)} \ln \frac{\kappa_L}{\kappa_0}$$

Problem 3 (5 points)

- DGD#4 Problem 19.42.
- Book example 19.7.
- Close to Problem 19.27, which had 97% correct answers.

Solutions:

Since two of the processes are adiabatic, no heat transfer occurs in those processes. Thus the heat transfer must occur along the isobaric processes.

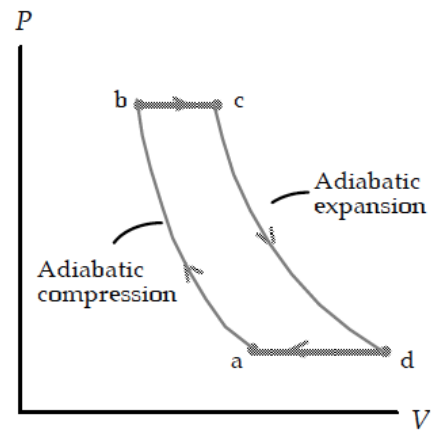
$$Q_H = Q_{bc} = nC_p(T_c - T_b) \quad ; \quad Q_L = Q_{da} = nC_p(T_d - T_a)$$

$$e = 1 - \frac{Q_L}{Q_H} = 1 - \frac{nC_p(T_d - T_a)}{nC_p(T_c - T_b)} = 1 - \frac{(T_d - T_a)}{(T_c - T_b)}$$

Use the ideal gas relationship, which says that $PV = nRT$.

$$e = 1 - \frac{(T_d - T_a)}{(T_c - T_b)} = 1 - \frac{\left(\frac{P_d V_d}{nR} - \frac{P_a V_a}{nR}\right)}{\left(\frac{P_c V_c}{nR} - \frac{P_b V_b}{nR}\right)} = 1 - \frac{(P_d V_d - P_a V_a)}{(P_c V_c - P_b V_b)}$$

$$= 1 - \frac{P_a(V_d - V_a)}{P_b(V_c - V_b)}$$



Because process ab is adiabatic, we have $P_a V_a^\gamma = P_b V_b^\gamma \rightarrow V_a = V_b \left(\frac{P_b}{P_a}\right)^{1/\gamma}$. Because process cd is

adiabatic, we have $P_b V_c^\gamma = P_a V_d^\gamma \rightarrow V_d = V_c \left(\frac{P_b}{P_a}\right)^{1/\gamma}$. Substitute these into the efficiency expression.

$$e = 1 - \frac{P_a(V_d - V_a)}{P_b(V_c - V_b)} = 1 - \frac{P_a \left(V_c \left(\frac{P_b}{P_a}\right)^{1/\gamma} - V_b \left(\frac{P_b}{P_a}\right)^{1/\gamma} \right)}{P_b(V_c - V_b)} = 1 - \frac{P_a \left(\frac{P_b}{P_a}\right)^{1/\gamma} (V_c - V_b)}{P_b(V_c - V_b)}$$

$$= 1 - \left(\frac{P_b}{P_a}\right)^{\frac{1}{\gamma} - 1} = \boxed{1 - \left(\frac{P_b}{P_a}\right)^{\frac{1-\gamma}{\gamma}}}$$

Problem 4 (5 points)

- DGD#4 Problem 20.53.
- Book example 20.6.

Solutions:

The equilibrium temperature is found using calorimetry arguments: The heat lost by the liquid is equal to the heat gained by the cup

$$m_{liq}c_{liq}(T_{liq} - T_{final}) = m_{cup}c_{cup}(T_{cup} - T_{final})$$

$$T_{final} < 100 \text{ } ^\circ\text{C} \quad (\sim 90 \text{ } ^\circ\text{C})$$

$$\begin{aligned} \Delta S &= \Delta S_{cup} + \Delta S_{liq} = \\ &= \int_{T_{cup}}^{T_{final}} \frac{dQ_{cup}}{T} + \int_{T_{liq}}^{T_{final}} \frac{dQ_{liq}}{T} = m_{cup}c_{cup} \int_{T_{cup}}^{T_{final}} \frac{dT}{T} + m_{liq}c_{liq} \int_{T_{liq}}^{T_{final}} \frac{dT}{T} \\ &= m_{cup}c_{cup} \ln \frac{T_{final}}{T_{cup}} + m_{liq}c_{liq} \ln \frac{T_{final}}{T_{liq}} \end{aligned}$$

Problem 5 (1 points)

(C)

This is the equation 20.21 of the book stating that the total entropy change during any reversible cycle is zero.

Problem 6 (1 points)

(C)

See example 19.3.

In any cycle $Q = W$, then the work done by the system is to the heat added to the system, which in this case is given by the area of the triangle: $+ 0.5 (0.02) (4,000) \text{ J}$