



**The University of Calgary  
Schulich School of Engineering  
ENGG 311: Engineering Thermodynamics  
Midterm Examination  
Fall 17**

**Open Textbook (or any equivalent textbook), Formula Sheet, Closed Notes, SSE,  
Calculators Allowed**

**Time Allowed: 90 minutes**

**Friday Nov. 3, 2017 – 17:30 h**

Name: Surname	Given Names	Lecture Section (Circle one)
		L01 (Dr. Husein); L02 (Dr. Moore)

**EXAMINATION INFORMATION**

1. Reference to **ONLY** the Textbook (or any equivalent textbook), Formula Sheet and property tables (8<sup>th</sup> Edition Cengel & Boles) is permitted during the examination. **CLOSED CLASS NOTES.** Additional material must not be inserted into the textbook, or the textbook will be confiscated.
2. **NO** electronic device, other than a calculator, may be brought into the exam. This includes telephones, cameras computers and music players of any kind.
3. SSE approved calculator is allowed.
4. Clearly show all steps involved in each calculation. Marks are awarded for both the method and the final answer
5. Please state any assumptions made wherever necessary.

**ADDITIONAL FORMULAS**

*Surface Area*

- Circle  $A = \frac{\pi D^2}{4}$
- Sphere  $A = \pi D^2$
- Cylinder  $A = \frac{\pi D^2}{2} + \pi D h$

*Volume*

- Sphere  $V = \frac{\pi D^3}{6}$
- Cylinder  $V = \frac{\pi D^2 h}{4}$
- Cone  $V = \frac{\pi D^2 h}{12}$

*Quadratic Polynomial*

- $ax^2 + bx + c = 0 \rightarrow x = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$

*Integrals*

- $\int_0^x \frac{dx}{1-x} = \ln\left(\frac{1}{1-x}\right)$
- $\int_0^x \frac{dx}{(1-x)^2} = \frac{x}{1-x}$
- $\int_0^x \frac{dx}{(1+ax)} = \frac{1}{a} \ln(1+ax)$

Question	Mark
1	/10
2	/10
3	/10
<b>Total</b>	<b>/30</b>

**Q1)** Air (assume ideal gas) is compressed from an initial state of 87.4 kPa and  $3.31 \times 10^{-4} \text{ m}^3$  to a final state of 165.6 kPa and  $2.14 \times 10^{-4} \text{ m}^3$ . During compression, the pressure varies with volume following  $PV^n = \text{const}$ . Calculate the heat interaction during this process (J). Assume  $C_v^0 = \text{const} = 0.718 \frac{\text{kJ}}{\text{kg}\cdot\text{K}}$ .

Air (ideal gas)  $\Rightarrow R = 0.2870 \left( \frac{\text{kJ}}{\text{kg}\cdot\text{K}} \right)$  "Table A-1"; Polytropic compression  
 State 1:  $P_1 = 87.4 \text{ kPa}$ ;  $V_1 = 3.31 \times 10^{-4} \text{ (m}^3)$   
 State 2:  $P_2 = 165.6 \text{ kPa}$ ;  $V_2 = 2.14 \times 10^{-4} \text{ (m}^3)$  }  $Q_{\text{net}} = ?$

Soln:

① Sys: Closed; ② W-F: Air (assume pure subs + ideal gas)  
 ③ Process: Polytropic compression; ④  $Q_{\text{net}}$  can be obtained from energy

balance:  $Q_{\text{net}} - W_{\text{net}} = m \Delta U$  "Stationary simple compressible sys"

$$\therefore W_{\text{net}} = W_b = \int_{V_1}^{V_2} P dV = \frac{P_2 V_2 - P_1 V_1}{1-n} \quad \text{"Polytropic process"}$$

$$\therefore P_1 V_1^n = \text{const} = P_2 V_2^n \Rightarrow \left( \frac{P_1}{P_2} \right) = \left( \frac{V_2}{V_1} \right)^n \quad \text{OR } n = \frac{\ln(P_2/P_1)}{\ln(V_2/V_1)}$$

$$\therefore n = \frac{\ln(87.4/165.6)}{\ln(2.14 \times 10^{-4} / 3.31 \times 10^{-4})} = -1.46$$

$$\therefore W_{\text{net}} = \frac{(165.6)(2.14 \times 10^{-4}) - (87.4)(3.31 \times 10^{-4})}{1-1.46} = -14.15 \text{ (J)} \quad (\because W_{\text{net}} < 0 \Rightarrow \text{Work done on the system})$$

$$\therefore m \Delta U = \int_{T_1}^{T_2} C_v dT \quad \text{assume } C_v = \text{const} = C_v @ 300\text{K} = 0.718 \left( \frac{\text{kJ}}{\text{kg}\cdot\text{K}} \right) \quad \text{"Table A-2"}$$

$$\therefore m_1 = m_2 \Rightarrow \frac{P_1 V_1}{RT_1} = \frac{P_2 V_2}{RT_2} \quad \therefore \Delta U = C_v (T_2 - T_1)$$

$$\Rightarrow \frac{T_2}{T_1} = \frac{P_2 V_2}{P_1 V_1} \quad \therefore m \Delta U = C_v \frac{P_1 V_1}{RT_1} (T_2 - T_1) \Rightarrow m \Delta U = C_v \frac{P_1 V_1}{RT_1} \left( \frac{P_2 V_2 T_1}{P_1 V_1} - T_1 \right)$$

$$\Rightarrow m \Delta U = C_v \left( \frac{P_2 V_2}{R} - \frac{P_1 V_1}{R} \right) \Rightarrow m \Delta U = \frac{C_v}{R} (P_2 V_2 - P_1 V_1)$$

$$\therefore m \Delta U = \frac{0.718 \text{ (kJ/kg}\cdot\text{K)}}{0.2870 \text{ (kJ/kg}\cdot\text{K)}} \left[ (165.6)(2.14 \times 10^{-4}) - (87.4)(3.31 \times 10^{-4}) \right] \text{ kJ} = 16.28 \text{ (J)}$$

$$\therefore Q_{\text{net}} - (-14.15) = 16.28 \Rightarrow Q_{\text{net}} = +2.13 \text{ J} \quad \leftarrow \text{added to the system}$$

Note  $Q_{\text{net}} \sim 0 \Rightarrow$  Process is adiabatic

**Q2)** Two tanks (A and B) are connected through a valve. The volume of tank A is 75 liters while that of tank B is 50 liters. Initially tank A contains carbon monoxide (CO) gas at a pressure of 1,000 kPa and a temperature of 30°C. Tank B is initially evacuated. The valve connecting the tanks is opened and the contents of the tanks are allowed to come to an equilibrium state at a temperature of 22°C. Assuming CO behaves as an ideal gas and that  $C_v^0 = 0.744 \frac{\text{kJ}}{\text{kg}\cdot\text{K}}$ :

- Calculate the mass of CO which is initially contained in tank A (kg).
- Calculate the masses of CO in tanks A and B after the valve is opened and the two tanks come to equilibrium (kg).
- Calculate the work done on the CO (kJ).
- Calculate the heat transfer to the CO as the two tanks come to equilibration (kJ).

Boundary CO  $M = 28.01$   $R = 0.2968 \frac{\text{kJ}}{\text{kg}\cdot\text{K}}$   
 $C_v^0 = 0.744 \frac{\text{kJ}}{\text{kg}\cdot\text{K}}$

Tank A:  $m_{A1} = \frac{P_{A1} V_{A1}}{R T_{A1}} = \frac{(1000 \text{ kPa}) \left(\frac{75}{1000} \text{ m}^3\right)}{0.2968 \frac{\text{kJ}}{\text{kg}\cdot\text{K}} (30+273) \text{ K}} = 0.834 \text{ kg} \leftarrow (a)$

Tank B:  $m_{B1} = 0$

After tanks connected, CO contained in both tanks

Mass balance  $m_{A1} + m_{B1} = m_{A2} + m_{B2}$

$V_{A1} + V_{B1} = V_{A2} + V_{B2} = (75+50) \text{ l} = 0.125 \text{ m}^3$

$\therefore$  At state 2:

$V_2 = 0.125 \text{ m}^3$   $T_2 = 22^\circ\text{C} = 295 \text{ K}$   $m_2 = m_{A2} + m_{B2} = m_{A1} = 0.834 \text{ kg}$

$\therefore P_2 = \frac{m_2 R T_2}{V_2} = \frac{(0.834 \text{ kg}) \left(0.2968 \frac{\text{kJ}}{\text{kg}\cdot\text{K}}\right) \times 295 \text{ K}}{0.125 \text{ m}^3}$

$P_2 = 584.2 \text{ kPa}$

b.  $m_{A2} = \frac{P_2 V_{A2}}{R T_2} = \frac{(584.2 \text{ kPa}) (0.075 \text{ m}^3)}{\left(0.2968 \frac{\text{kJ}}{\text{kg}\cdot\text{K}}\right) (295 \text{ K})} = 0.500 \text{ kg}$

$m_{B2} = \frac{P_2 V_{B2}}{R T_2} = \frac{(584.2 \text{ kPa}) (0.050 \text{ m}^3)}{0.2968 \frac{\text{kJ}}{\text{kg}\cdot\text{K}} \times 295 \text{ K}} = 0.334 \text{ kg}$

$\underline{0.834 \text{ kg}}$

c.  $W = \int P dV = 0$  as  $dV = 0$

(d)  $Q - \cancel{W} = m_{A2} u_{A2} + m_{B2} u_{B2} - m_{A1} u_{A1} - m_{B1} u_{B1}$

$= m_{A2} u_{A2} - m_{A1} u_{A1}$

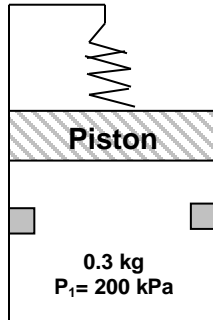
$= m_{A1} [u_2 - u_1] = m_{A1} C_v^0 [T_2 - T_1]$

$Q = (0.834 \text{ kg}) \left(0.744 \frac{\text{kJ}}{\text{kg}\cdot\text{K}}\right) [22 - 30]^\circ\text{C} \times \frac{\text{K}}{^\circ\text{C}}$

$= -4.96 \text{ kJ}$  (Heat transferred from tanks).

**Q3)** A spring-loaded frictionless piston-cylinder device contains 0.30 kg of superheated steam at 200 kPa. It is cooled to a final temperature of 20°C. At the final state, 0.29 kg existed in the liquid phase. If the final volume is 80% of its initial volume and the stops set at 0.20 m<sup>3</sup>, calculate:

- The temperature of the system at the initial state (°C)
- The boundary work involved in the process (kJ)
- The heat interaction with the surrounding (kJ)
- Sketch the process on a P-v diagram.



$m = 0.3 \text{ kg}$ ; Spring attached throughout; State 1:  $P_1 = 200 \text{ kPa}$   
 State final:  $T_f = 20^\circ\text{C}$ ,  $m_L = 0.29 \text{ kg}$   $\Rightarrow x_f = \frac{0.3 - 0.29}{0.3} = 0.033$ ;  $V_f = 0.8 V_1$   
 $V$  corresponding to the stops =  $0.2 \text{ m}^3$   
 a)  $T_1 = ?$ ; b)  $W_b = ?$ ; c)  $Q = ?$ ; d) Sketch P-v diagram  
 Soln:  
 ① Sys: closed; ② W.F.: Water (use prop. tables); ③ Process: Linear-spring + could be isochoric  
 1st need to establish if piston hits the stops @ the final stage or not.  
 @ the final state  $\Rightarrow$  l-v mix @  $T_f = 20^\circ\text{C}$  &  $x_f = 0.033$ : Table A-4:  
 $v_f @ 20^\circ\text{C} = 0.001002 \text{ (m}^3/\text{kg)}$  &  $v_g = 57.762 \text{ (m}^3/\text{kg})$   $\Rightarrow v_{\text{final}} = 0.001002 + 0.033(57.762 - 0.001002) = 1.926 \text{ (m}^3/\text{kg)}$   
 $V_{\text{final}} = m v_{\text{final}} = (0.3 \text{ kg})(1.926 \text{ m}^3/\text{kg}) = 0.578 \text{ (m}^3)$   
 $V_{\text{final}} > V$  corresponding to stops  $\Rightarrow$  Piston does not hit the stops.  
 $\Rightarrow$  P varies linearly with v  
 $V_1 = \frac{V_{\text{final}}}{0.8} \Rightarrow V_1 = 0.722 \text{ (m}^3)$  &  $v_1 = \frac{0.722 \text{ (m}^3)}{0.3 \text{ (kg)}} = 2.408 \text{ (m}^3/\text{kg)}$   
 @  $P_1 = 200 \text{ kPa}$ ,  $v_1 > v_g (0.88578 \text{ m}^3/\text{kg})$   $\Rightarrow$  State 1 Superheated vapor  
 $T_1 = 700 + \frac{700 - 800}{2.24434 - 2.47550} (2.408 - 2.24434) = 770.8^\circ\text{C}$   
 b) Spring engage + piston did not reach the stops  $\Rightarrow$   
 $W_b = \int_{P_1}^{P_2} P dv = \frac{P_1 + P_2}{2} (v_2 - v_1)$   $\Rightarrow P_2 = P_{20^\circ\text{C}} = 2.339 \text{ (kPa)}$   
 $W_b = \frac{(200 + 2.339) \text{ kPa}}{2} (0.578 - 0.722) \text{ m}^3 = -14.57 \text{ kJ}$  (work done on the sys)  
 c) 1st Law:  $Q_{\text{net}} - W_{\text{net}} = m(u_2 - u_1)$  Stationary closed system  
 $u_1 = 3479.9 + \frac{(3479.9 - 3664.7)}{700 - 800} (770.8 - 700) = 3,610.7 \text{ (kJ/kg)}$  (Table A-6)  
 $u_2 = 83.913 + 0.033(2318.4) = 160.4 \text{ (kJ/kg)}$   
 $Q_{\text{net}} - (-14.57 \text{ kJ}) = 0.3 \text{ (kg)} (160.4 - 3,610.7) \text{ (kJ/kg)} \Rightarrow Q_{\text{net}} = -1050 \text{ kJ}$  (lost to the surrounding)

