

ASSIGNMENT 6 Solutions**PROBLEM 1 (25 points)**

For the elbow duct shown in Figure 1 below, SAE 30 oil ($\gamma = 8720\text{N/m}^3$) at 20°C enters section 1 at 350N/s , and exits at section 2. Assuming steady incompressible flow, compute the force, and its direction, of the oil on the elbow due to momentum change only (no pressure change or friction effects).

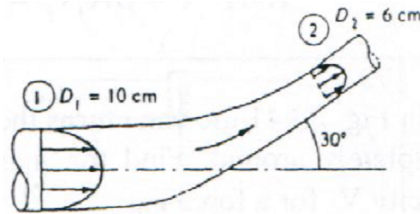


Figure 1

Solution:

$$\gamma = 8790\text{N/m}^3$$

$$\text{thus } Q = \frac{350}{8720} = 0.0401 \frac{\text{m}^3}{\text{s}}$$

$$Q = 0.0401 \frac{\text{m}^3}{\text{s}} = \pi(0.05)^2 V_{av,1} = \pi(0.03)^2 V_{av,2}$$

$$V_{av,1} = 5.11\text{m/s}$$

$$V_{av,2} = 14.2\text{m/s}$$

$$F_x = \beta_2 \rho A_2 V_{av,2}^2 \cos \theta - \beta_1 \rho A_1 V_{av,1}^2; \quad F_y = \beta_2 \rho A_2 V_{av,2}^2 \sin \theta$$

(a) Neglecting Momentum – Flux Correction Factors ($\beta = 1$)

$$F_x = (890)\pi(0.03)^2(14.2)^2 \cos 30^\circ - (890)\pi(0.05)^2(5.11)^2 = 439 - 183 \approx \mathbf{256\text{ N}}$$

$$F_y = (890)(\pi)(0.03)^2(14.2)^2 \sin(30) = \mathbf{254\text{ N}}$$

PROBLEM 2 (25 points)

Water at 20°C flows through a 5-cm-diameter pipe which has a 180° vertical bend, as shown in Figure 2. The total length of the pipe between flanges 1 and 2 is 75cm. When the weight flow rate is 230N/s, $p_1=165$ kPa, and $p_2= 134$ kPa. Neglecting pipe weight, determine the total force which the flanges must withstand for this flow.

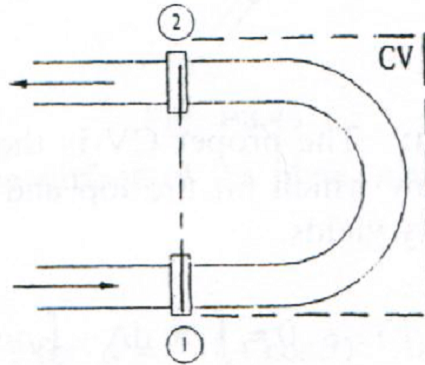


Figure 2

Solution: Let the CV cut through the flanges and surround the pipe bend. The mass flow rate is $(230 \text{ N/s})/(9.81 \text{ m/s}^2) = 23.45 \text{ kg/s}$. The volume flow rate is $Q = 230/9790 = 0.0235 \text{ m}^3/\text{s}$. Then the pipe inlet and exit velocities are the same magnitude:

$$V_1 = V_2 = V = Q/A = \frac{0.0235 \text{ m}^3/\text{s}}{(\pi/4)(0.05 \text{ m})^2} \approx 12.0 \frac{\text{m}}{\text{s}}$$

Subtract p_a everywhere, so only p_1 and p_2 are non-zero. The horizontal force balance is:

$$\begin{aligned} \sum F_x &= F_{x,\text{flange}} + (p_1 - p_a)A_1 + (p_2 - p_a)A_2 = \dot{m}_2 u_2 - \dot{m}_1 u_1 \\ &= F_{x,\text{fl}} + (64000) \frac{\pi}{4} (0.05)^2 + (33000) \frac{\pi}{4} (0.05)^2 = (23.45)(-12.0 - 12.0 \text{ m/s}) \\ \text{or: } F_{x,\text{flange}} &= -126 - 65 - 561 \approx \mathbf{-750 \text{ N}} \quad \text{Ans.} \end{aligned}$$

PROBLEM 3 (20 points)

In Figure 3 a perfectly balanced 700-N weight and level platform are supported by a steady water jet. What is the proper jet velocity?

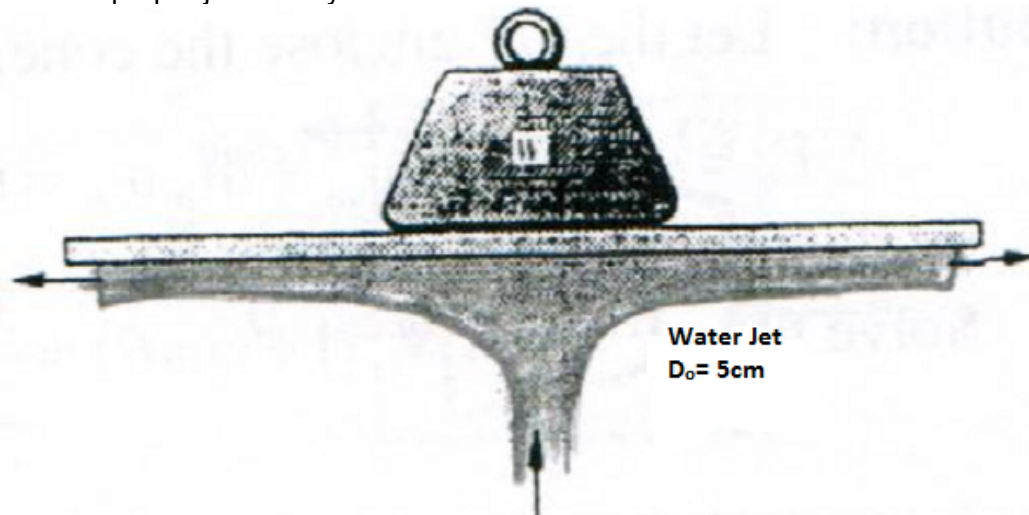


Figure 3

Solution:

For a CV around surrounding the weight, platform, and jet, vertical forces yield:

$$\sum F_y = -W = \dot{m}_{\text{left}} V_{\text{left}} + \dot{m}_{\text{right}} V_{\text{right}} - \dot{m}_o V_o = 0 + 0 - (\rho_o A_o V_o) V_o$$

$$\text{Thus } W = 700 \text{ N} = (998) \frac{\pi}{4} (0.05)^2 V_o^2, \text{ solve for } V_o = V_{\text{jet}} = 18.9 \frac{\text{m}}{\text{s}} \text{ Ans.}$$

PROBLEM 4 (30 points)

A liquid jet of density ρ and area A strikes a block and splits into two jets, as shown in Fig. 4. All three jets have the same velocity V . The upper jet exits at angle θ and area αA , the lower jet turns down at 90° and area $(1 - \alpha)A$. (a) Derive a formula for the forces (F_x, F_y) required to support the block against momentum changes. (b) Show that $F_y = 0$ only if $\alpha \geq 0.5$. (c) Find the values of α and θ for which both F_x and F_y are zero.

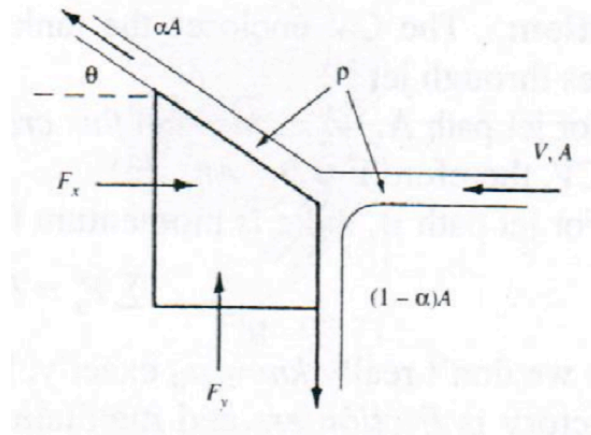


Figure 4

Solution: (a) Set up the x - and y -momentum relations:

$$\sum F_x = F_x = \alpha \dot{m}(-V \cos \theta) - \dot{m}(-V) \quad \text{where } \dot{m} = \rho AV \text{ of the inlet jet}$$

$$\sum F_y = F_y = \alpha \dot{m} V \sin \theta + (1 - \alpha) \dot{m}(-V)$$

Clean this up for the final result:

$$F_x = \dot{m} V (1 - \alpha \cos \theta)$$

$$F_y = \dot{m} V (\alpha \sin \theta + \alpha - 1) \quad \text{Ans. (a)}$$

(b) Examining F_y above, we see that it can be zero only when,

$$\sin \theta = \frac{1 - \alpha}{\alpha}$$

But this makes no sense if $\alpha < 0.5$, hence $F_y = 0$ only if $\alpha \geq 0.5$. Ans. (b)

(c) Examining F_x we see that it can be zero only in $\cos \theta = 1/\alpha$, which makes no sense unless $\alpha = 1$, $\theta = 0^\circ$. This situation also makes $F_y = 0$ above ($\sin \theta = 0$). Therefore the only scenario for which both forces are zero is the trivial case for which all the flow goes horizontally across a flat block:

$$F_x = F_y = 0 \text{ only if: } \alpha = 1, \theta = 0^\circ$$