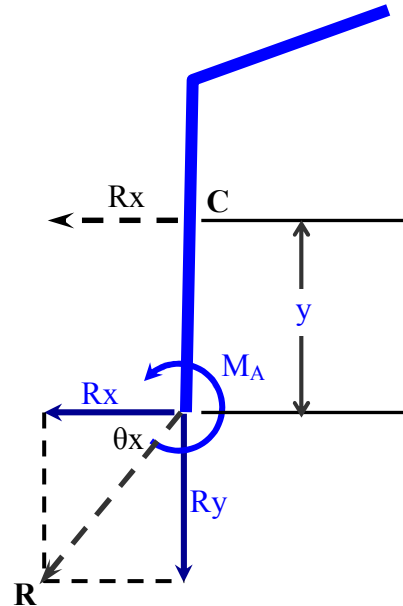
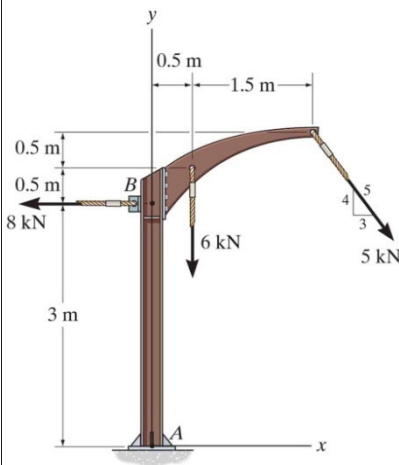


Problem 1 (15 Marks) F 4.34 (p 178)

For the system shown, determine

- The rectangular components of the resultant force, R .
- The magnitude of the resultant force, R .
- The orientation, θ_x , of the resultant force, with respect to the horizontal axis.
- The intersection, v . of the line of action of the resultant with the y -axis

**Force Resultant at A**

$$R_x = -8 + \left(\frac{3}{5}\right)5 = -5 \text{ kN} \leftarrow \text{(a)}$$

$$R_y = -6 - \left(\frac{4}{5}\right)5 = -10 \text{ kN} \leftarrow \text{(a)}$$

$$R = \sqrt{R_x^2 + R_y^2} = 11.2 \text{ kN} \leftarrow \text{(b)}$$

$$\theta_x = \tan^{-1} \left[\frac{R_y}{R_x} \right] = \tan^{-1} [2] = 63.4^\circ \leftarrow \text{(c)}$$

Moment Resultant at A

$$M_A = 8(3) - 6(0.5) - \left(\frac{3}{5}\right)5(3 + 0.5 + 0.5) - \left(\frac{4}{5}\right)5(0.5 + 1.5)$$

$$M_A = 24 - 3 - 12 - 8 = 1 \text{ kN.m}$$

Move resultant force from A to C

(1) Slide R_y upwards along its vertical line of action.

[This does not produce any moment]

(2) Slide R_x upwards, parallel to its line of action

[This "carries" a moment = $R_x(y)$]

The resultant moment at C should be zero

$$M_C = -R_x(y) + R_y(0) + M_A = 0$$

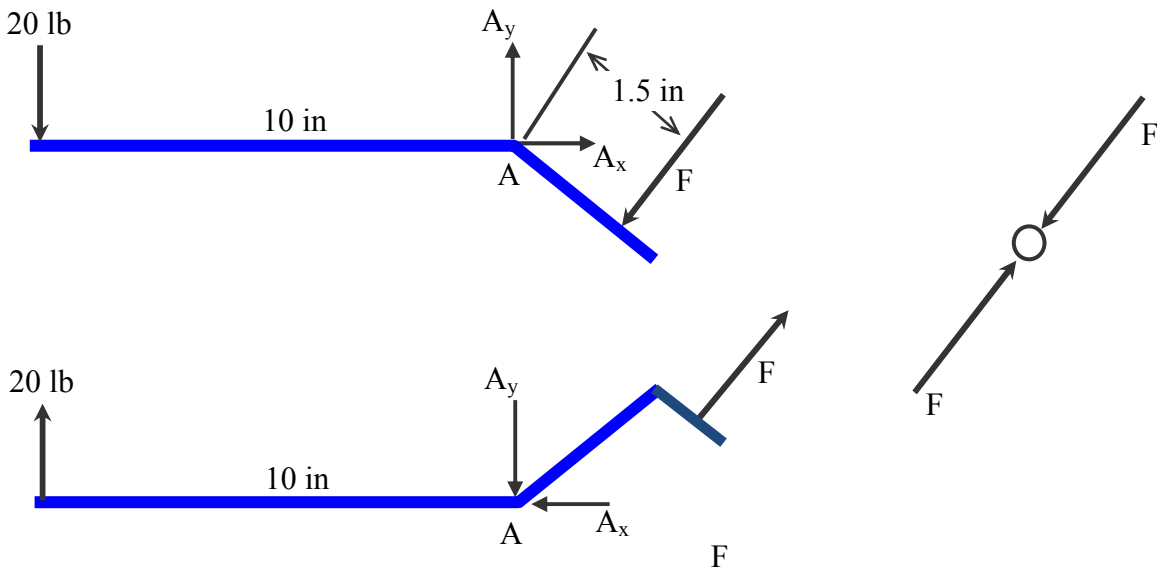
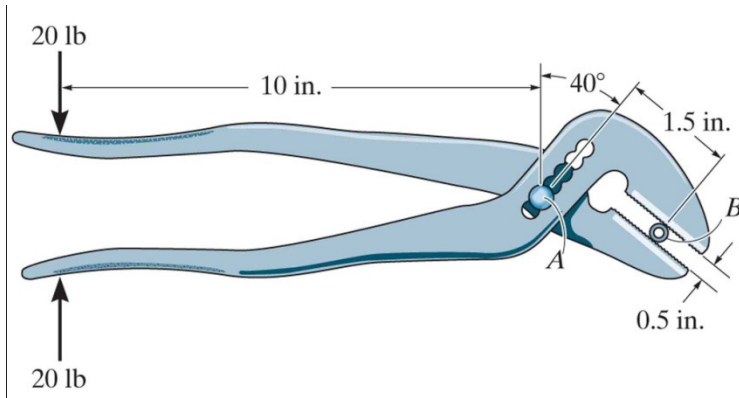
Solve for y

$$y = \frac{M_A}{R_x} = \frac{1 \text{ kN.m}}{5 \text{ kN}} = 0.200 \text{ m} \quad \text{(d)}$$

Problem 2(10Marks) 6.127 (p326)

For the pliers shown:

- (a) Draw a free body diagram of the top handle
- (b) Draw a free body diagram of the bottom handle
- (c) Draw a free body diagram of the smooth pipe at B
- (d) Determine the clamping force exerted on the smooth pipe at B if a force of 20 lb is applied to the handles of the pliers. The pliers are pinned at A.



From the FBD of the top handle:

$$\Sigma M_A = 0$$

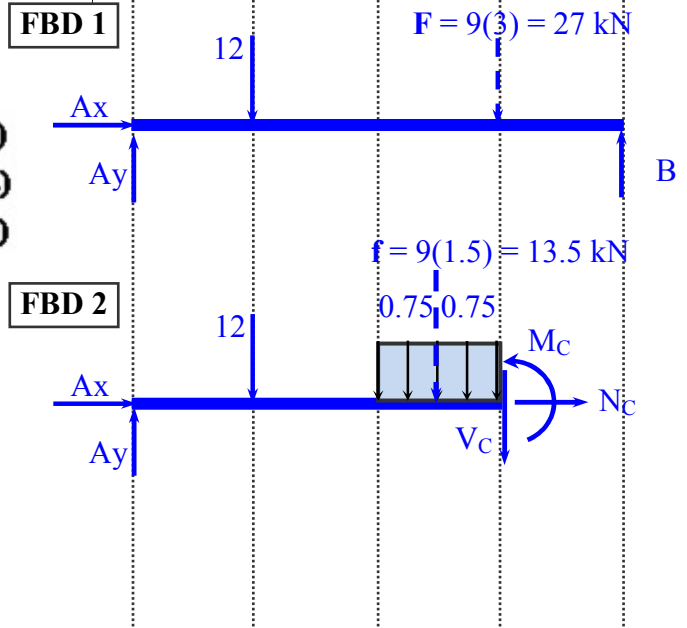
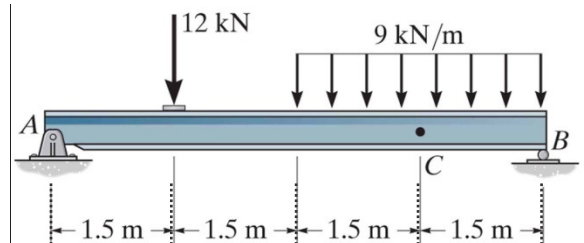
$$20(10) - F(1.5) = 0$$

$$F = \frac{200}{1.5} = 133 \text{ lb}$$

Problem 3(15Marks) F 7.4 (p337)

For the beam shown, determine:

- (a) Reactions at A and B
- (b) Internal forces at point C
 - Axial Force, N_c
 - Shear Force, V_c
 - Bending Moment, M_c



(a) Reactions. From FBD 1

$$\Sigma F_x = 0 \rightarrow A_x = 0 \quad \text{---(1)}$$

$$\Sigma F_y = 0 \rightarrow A_y - 12 - F + B = 0 \quad \text{---(2)}$$

$$\Sigma M_B = 0 \rightarrow -A_y(6) + 12(4.5) + 27(1.5) = 0 \quad \text{---(3)}$$

$$\text{From (3)} \quad A_y = \frac{94.5}{6} = 15.75 = 15.8 \text{ kN}$$

$$\text{From (2)} \quad B = 27 + 12 - 15.75 = 23.25 \text{ kN}$$

(b) Internal forces at C. From FBD 2

$$\Sigma F_x = 0 \rightarrow N_c = 0$$

$$\Sigma F_y = 0 \rightarrow 15.75 - 12 - 13.5 - V_c = 0$$

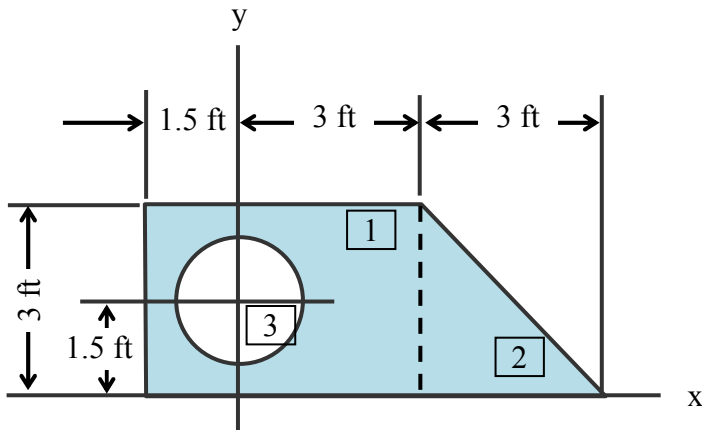
$$V_c = -9.75 = 9.8 \text{ kN} \uparrow$$

$$\Sigma M_C = 0 \rightarrow -15.75(4.5) + 12(3) + 13.5(0.75) + M_c = 0$$

$$M_c = +24.75 = 24.8 \text{ kN.m}$$

Problem 4(20Marks) 9.60 (p 479)

A 2 ft diameter hole is punched out of the trapezoidal plate shown. Locate the coordinates \bar{x} and \bar{y} for the centre of gravity of this composite.



Component	A_i (ft ²)	x_i (ft)	y_i (ft)	$A_i x_i$ (ft ³)	$A_i y_i$ (ft ³)
Rectangle (1)	$3(4.5)=13.5$	$0.5(4.5)-1.5=0.75$	$0.5(3)=1.5$	$(13.5)(0.75)$	$(13.5)(1.5)$
Triangle (2)	$0.5(3)(3)=4.5$	$3+3/3=4$	$3/3=1$	$(4.5)(4)$	$(4.5)(1)$
Circle (3)	$-\Pi(1)^2=-\Pi$	0	1.5	0	-1.5Π
	$\Sigma=14.858$			$\Sigma=28.125$	$\Sigma=20.038$

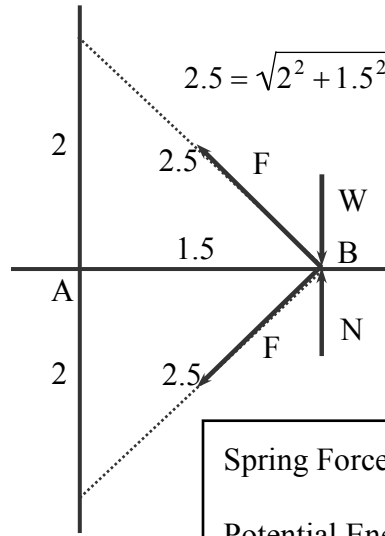
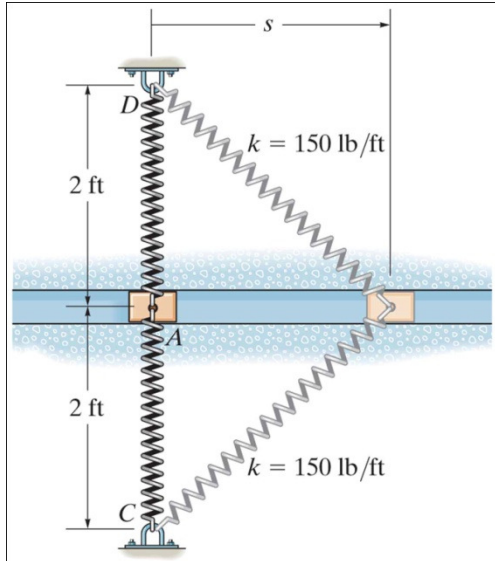
Centre of Gravity:

$$x = 28.125 / 14.858 = 1.89 \text{ ft}$$

$$y = 20.038 / 14.858 = 1.35 \text{ ft.}$$

Problem 5(20Marks) 14.79 (p 211)

Block A has a weight of 1.5 lb and slides in the smooth horizontal slot. If the block is drawn back to $s = 1.5$ ft and released from rest, determine its speed at the instant $s = 0$. Each of the two springs has a stiffness of $k = 150$ lb/ft and an unstretched length of 0.5 ft.



Length units in ft.

Spring Force: $F = k \Delta L$
 Potential Energy of Spring: $V = \frac{1}{2} k \Delta L^2$

Conservation of Energy $\rightarrow T_B + V_B = T_A + V_A$

$$T_B = \frac{1}{2} m v_B^2 = \frac{1}{2} m 0^2 = 0$$

$$V_B = 2 \left[\frac{1}{2} k \Delta L^2 \right] = k \Delta L^2 \rightarrow \text{Two Springs}$$

$$V_B = 150(2.5 - 0.5)^2 = 600J$$

$$T_A = \frac{1}{2} m v_A^2 = \frac{1}{2} \left(\frac{W}{g} \right) v_A^2 = \frac{1}{2} \left(\frac{1.5}{32.2} \right) v_A^2 = \frac{v_A^2}{42.933}$$

$$V_A = k \Delta L^2 = 150(2 - 0.5)^2 = 337.5J$$

Conservation of Energy

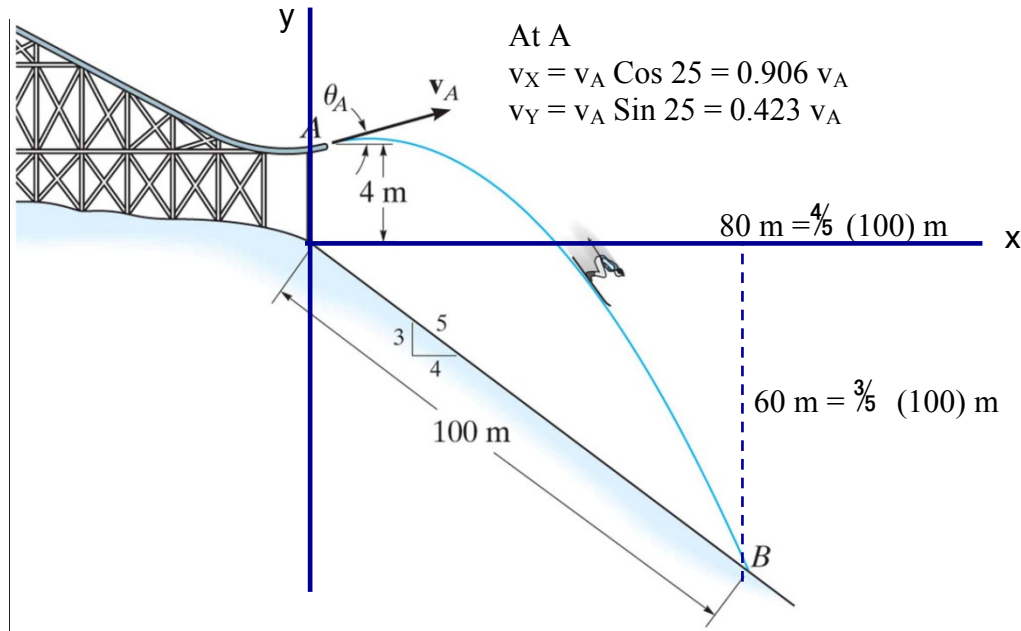
$$0 + 600 = \frac{v_A^2}{42.933} + 337.5$$

$$v_A^2 = (600 - 337.5)42.933 = 11269.9$$

$$v_A = 106.2 \text{ ft/s} \quad \leftarrow$$

Problem 6(20Marks) 12.110 (p 52)

The skier leaves the ramp A at an angle $\theta_A = 25^\circ$ with the horizontal. If she strikes the ground at B, determine her initial speed v_A and the time of flight t_{AB} .



Horizontal Motion

$$x_B = v_x t_B$$

$$80 = 0.906 v_A t_B$$

$$t_B = \frac{88.3}{v_A}$$

Vertical Motion

$$y = y_0 + v_y t_B + \frac{1}{2} a_c t_B^2$$

$$-60 = 4 + 0.423 v_A t_B + \frac{1}{2} (-9.8) t_B^2$$

Substitute $t_B = \frac{88.3}{v_A}$

$$0 = 64 + 0.423 v_A \left(\frac{88.3}{v_A} \right) - 4.9 \left(\frac{88.3}{v_A} \right)^2$$

Solving

$$v_A = 19.4 \text{ m/s} \quad \triangleleft$$

$$t_B = \frac{88.3}{v_A} = \frac{88.3}{19.4} = 4.6 \text{ s} \quad \triangleleft$$