

**MCG 3130
DGD 6 (KINOSTATIC)**

Nov 7th 2014.

Pls Note:

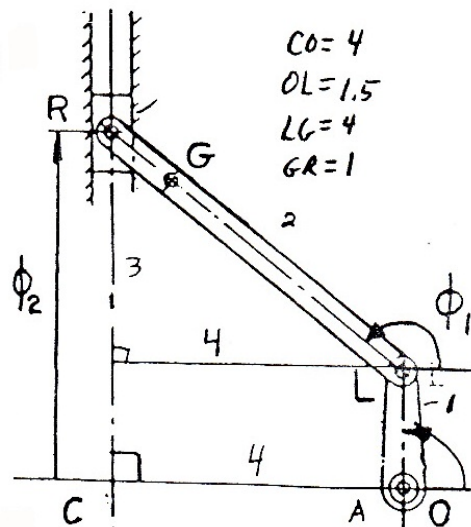
First Question is the **BOARD PROBLEM QUESTION**

Second question, will be the **ASSIGNED QUESTIONS To Students**

BOARD QUESTION 1:

For the mechanism of Fig. 3 OL is a crank with fixed axis at O, and LR is a connecting rod pinned to a slider at R which slides in fixed vertical guides. $CO=4$ cm, $OL=1.5$ cm, $LG=4$ cm, and $GR=1$ cm. At the instant of consideration OL is vertical and $CR=4.5$ cm. Crank OL is balanced. Link LR, with center of mass at G, has a mass of $m_2 = 0.2\text{kg}$, and a mass moment of inertia about G of $I_{zz} = 0.1\text{kg}\cdot\text{cm}^2$. The slider R has a mass of $m_3 = 0.1\text{kg}$. A kinematic analysis has provided the following information at this instant: $\dot{\phi}_1 = 2\text{rad/s}$ counterclockwise (const.), $\dot{\phi}_2 = -1\text{ rad/s}$, $\ddot{\phi}_1 = 1.33\text{ rad/s}^2$, $\dot{\phi}_2 = 4\text{ cm/s}$, $\ddot{\phi}_2 = -14.3\text{ cm/s}^2$. Ignoring effects of friction and gravity determine for this instant

- the D'Alembert force sets for the connecting rod LR and the slider R,
- the necessary driving torque T on link OL,
- the forces at the pins O, L, and R.

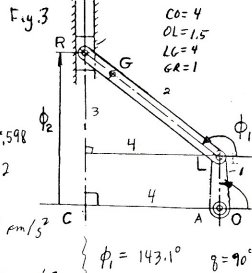


Solution:

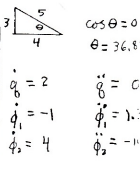
SOLN:

(a) D'Alembert force sets links 2 & 3

$$\begin{aligned}
 x_{G2} &= 1.5 \cos \theta + 4 \cos \phi_1 \\
 y_{G2} &= 1.5 \sin \theta + 4 \sin \phi_1 \\
 \dot{x}_{G2} &= -1.5 \sin \theta \dot{\theta} - 4 \sin \phi_1 \dot{\phi}_1 = -1.5(1)(2) - 4(1.6004)(-1) = -5.98 \\
 \dot{y}_{G2} &= 1.5 \cos \theta \dot{\theta} + 4 \cos \phi_1 \dot{\phi}_1 = 1.5(0) + 4(-1.8)(-1) = 3.2 \\
 \ddot{x}_{G2} &= -1.5 \sin \theta \ddot{\theta} - 1.5 \cos \theta \dot{\theta}^2 - 4 \sin \phi_1 \ddot{\phi}_1 - 4 \cos \phi_1 \dot{\phi}_1^2 \\
 &= -1.5(1)0 - 1.5(0) - 4(1.6004)1.33 - 4(-1.8)(1) = -0.008 \text{ cm/s}^2 \\
 \ddot{y}_{G2} &= 1.5 \cos \theta \ddot{\theta} - 1.5 \sin \theta \dot{\theta}^2 + 4 \cos \phi_1 \ddot{\phi}_1 - 4 \sin \phi_1 \dot{\phi}_1^2 \\
 &= 1.5(0) - 1.5(1)4 + 4(-1.8)1.33 - 4(1.6004)(1) = -12.7 \text{ cm/s}^2
 \end{aligned}$$



$$\begin{aligned}
 \underline{F}_{D2} &= -m_2 \underline{a}_{G2} = -0.2 (\ddot{x}_{G2} \underline{i} + \ddot{y}_{G2} \underline{j}) = (-0.0016 \underline{i} + 2.54 \underline{j}) \times 10^{-2} \text{ N} \\
 \underline{C}_{D2} &= -I_{G2} \underline{\alpha}_2 \underline{k} = -0.11 \times 10^{-4} (1.33) \underline{k} = -0.133 \times 10^{-4} \underline{k} \text{ Nm} \\
 \underline{F}_{D3} &= -m_3 \underline{a}_{G3} = -0.1 (-14.3 \underline{j}) \times 10^{-2} = 1.43 \times 10^{-2} \underline{j} \text{ N} \\
 \underline{C}_{D3} &= 0
 \end{aligned}$$



(b) P.V.W $\sum \underline{T} \cdot \underline{\omega} + \sum \underline{F} \cdot \underline{v} = 0$

$$\begin{aligned}
 T_1 \underline{k} \cdot \dot{\theta} \underline{k} + C_{D2} \cdot \dot{\phi}_1 \underline{k} + \underline{F}_{D2} \cdot (\dot{x}_{G2} \underline{i} + \dot{y}_{G2} \underline{j}) + \underline{F}_{D3} \cdot \dot{\phi}_2 \underline{j} &= 0 \\
 \text{i.e. } T_1 (2) + (-0.133 \times 10^{-4})(-1) + (0 \underline{i} + 2.54 \underline{j}) \cdot (-1.6 \underline{i} + 3.2 \underline{j}) \times 10^{-4} + (1.43) (4) \times 10^{-4} &= 0 \\
 T_1 = -\frac{1}{2} [0.133 + 8.13 + 5.72] \times 10^{-4} & \quad \underline{T}_1 = -7 \times 10^{-4} \text{ Nm}
 \end{aligned}$$

(c) $\sum M_L = 0$

$$\begin{aligned}
 3F_{Rx} + 4F_{D3} + 2.4F_{2x} + 3.2F_{2y} &= 0 \\
 F_{Rx} &= -\frac{10}{3} [4(1.43) + 2.4(0) + 3.2(2.54)] = -4.65 \times 10^{-2} \\
 -F_{Lx} + F_{2x} + F_{Ax} &= 0 \quad F_{Lx} = -4.65 \times 10^{-2} \\
 -F_{Ly} + F_{2y} + F_{D3} &= 0 \quad F_{Ly} = 3.97 \times 10^{-2} \\
 F_{Ly} &= 2.54 + 1.43
 \end{aligned}$$

Check $(F_{Rx})(0.1) = 0.43 \times 10^{-4} \approx 7 \times 10^{-4}$

ASSIGNED QUESTION 2:

The link lengths and constant distances for the mechanism of Fig. 2 are given by $r_1 = 3.4\text{cm}$, $r_2 = 6.0\text{cm}$, $l_1 = 3.25\text{cm}$, $l_2 = 3.72\text{cm}$, and $c_1 = 4.2\text{cm}$. Link BCD has angle $\text{CBD} = 34^\circ$, and the side DC vertical at the instant shown. At this instant the mechanism position is given by $q = 15^\circ$, $\phi_1 = 299^\circ$, $\phi_2 = 6.2\text{cm}$. Link 1 is a uniform member of mass $m_1 = 0.1\text{kg}$. Link 2, with center of mass at D, has a mass of $m_2 = 0.2\text{kg}$, and a mass moment of inertia about D of $I_{2z} = 0.1\text{kg}\cdot\text{cm}^2$. The slider (link 3) has a mass of $m_3 = 0.2\text{kg}$. A kinematic analysis has provided the following information for the motion of the linkage: $\dot{q} = 10\text{rad/s}$ CCW (cnst.), $\dot{\phi}_1 = -11.3\text{ rad/s}$, $\ddot{\phi}_1 = -200\text{ rad/s}^2$, $\dot{\phi}_2 = 68.1\text{ cm/s}$ (to left), $\ddot{\phi}_2 = 1750\text{ cm/s}^2$ (to left). Ignoring effects of friction and gravity determine for this instant

- the D'Alembert force set corresponding to each of the three links,
- the necessary driving torque T_1 on link 1.
- the forces at the pins A, B, and C.

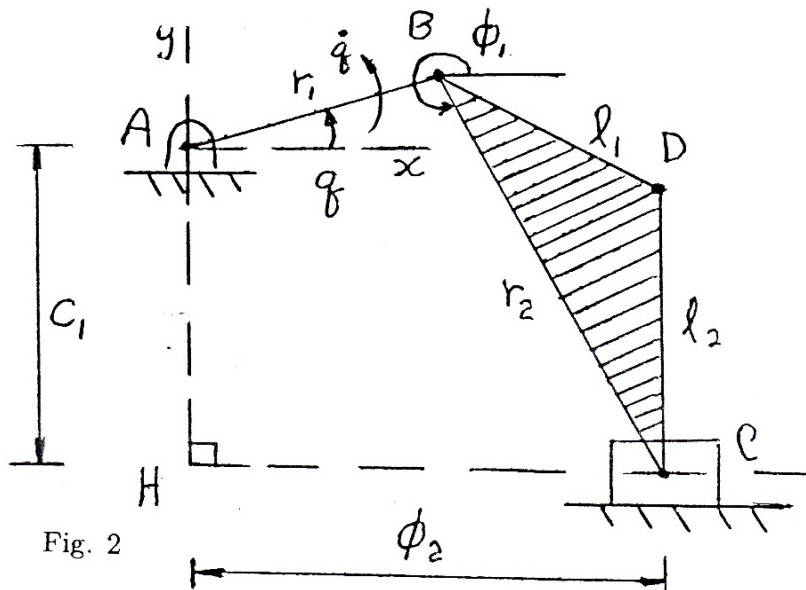


Fig. 2

Solution:

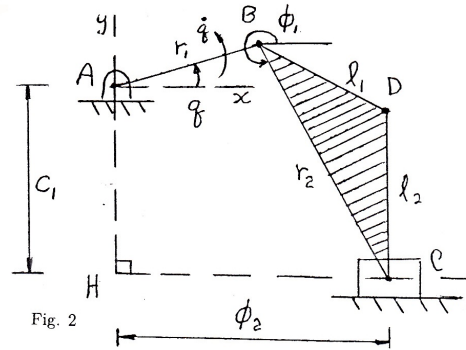


Fig. 2

(a) D'Alembert force sets

$$\begin{aligned} \ddot{q}_1 &= -\dot{\omega}^2 \frac{AB}{2} \hat{e}_{AB} \\ &= -172 \cos 15^\circ \hat{i} - 172 \sin 15^\circ \hat{j} \\ &= -166 \hat{i} - 44.5 \hat{j} \text{ cm/s}^2 \end{aligned}$$

$$\begin{aligned} \ddot{a}_{02} &= \ddot{q}_0 = \ddot{q}_C + \ddot{a}_{D/C} \\ &= -1750 \hat{i} + \dot{\phi}_1^2 l_2 (\hat{i}) + \dot{\phi}_1^2 l_2 (-\hat{j}) \\ &= -1006 \hat{i} - 475 \hat{j} \end{aligned}$$

Link 1 $F_{01} = (.166 \hat{i} + .045 \hat{j}) N$
 $C_{01} = 0$

Link 2 $F_{02} = (2.01 \hat{i} + 0.95 \hat{j}) N$
 $C_{02} = +200 \times 10^{-5} \frac{k}{Nm} = -0.002 Nm$

Link 3 $F_{03} = +3.5 \hat{i} N$
 $C_{03} = 0$

- $r_1 = 3.4 \quad \angle CBO = 3\pi/4$
- $r_2 = 6.0 \quad \angle HCD = 90^\circ$
- $l_1 = 3.25 \quad \delta = 15^\circ$
- $l_2 = 3.72 \quad \phi_1 = 299^\circ$
- $c_1 = 4.2 \quad \phi_2 = 6.2$
- $m_1 = 0.1 \quad I$
- $m_2 = 0.2 \quad I_{30} = 0.1$
- $m_3 = 0.2$
- $\dot{q}_1 = 10 \quad \dot{q}_2 = 0$
- $\dot{\phi}_1 = -11.3 \quad \dot{\phi}_2 = -200$
- $\dot{\phi}_2 = 68.1 \rightarrow \dot{\phi}_2 = 1750 \rightarrow$

(b) Driving torque T_1 - V.W. eqn

$$\underline{v}_{C1} = \underline{\omega}_1 \times \frac{AB}{2} = 10 \hat{k} \times 1.7 (0.966 \hat{i} + 0.259 \hat{j})$$

$$= -4.40 \hat{i} + 16.42 \hat{j} \text{ cm/s}$$

$$\underline{v}_{C2} = \underline{v}_D = \underline{v}_C + \underline{v}_{D/C} = -68.1 \hat{i} + 11.3 (3.72) \hat{i}$$

$$= -26.1 \hat{i} \text{ cm/s}$$

$$\underline{v}_{C3} = -68.1 \hat{i} \text{ cm/s}$$

Principle of Virtual Work $\sum T_i \cdot \dot{\omega}_i + \sum F_i \cdot \dot{v}_i = 0$

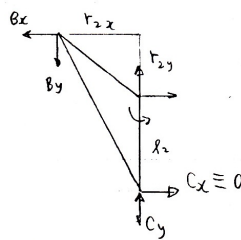
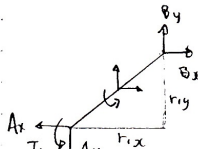
$$T_1 (10) + (0.002) (-11.3) + (.166 \hat{i} + .045 \hat{j}) \cdot (-4.40 \hat{i} + 16.42 \hat{j}) \times 10^{-2}$$

$$+ 10^2 [(2.01 \hat{i} + 0.95 \hat{j}) \cdot (-26.1 \hat{i}) + 3.5 \hat{i} \cdot (-68.1) \hat{i}] = 0$$

$$10 T_1 + 10^{-2} [-22.6 - 0.73 + 0.74 - 52.5 - 238.35] = 0$$

$$T_1 = 0.313 \text{ Nm}$$

(c)



Procedure

link 1 $\sum M_A = 0 \quad B_x, B_y$
 2 $\sum M_C = 0 \quad B_x, B_y$

link 2 $\sum F = 0 \quad A_x, A_y$
 1 $\sum F = 0 \quad C_y$

$$C_x \equiv 0$$